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Large deflection of a circular plate under non-uniform load pertaining to Aleph-Functions

Research Article

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Abstract: The main object of the present paper is to obtain the large deflection and bending stresses for a clamped circular plate under non-uniform load by using Berger's approximate method. The load shape considered here is an arbitrary function $p(x)$ involving Jacobi polynomial, Fox-Wright function and Aleph-functions. The small deflection case is also considered as a particular case of large deflection. The obtained results of this paper provide an extension of the results given by the literature.

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Keywords: \aleph -function • I-function • Jacobi polynomial • Fox-Wright Function

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1. Introduction

A lot of research work has been recently come up on the study and development of a function that is more general than I-function and Fox's H-function, known as the Aleph (\aleph)-function. The Aleph (\aleph)-function introduced by *Süddland* et al. [1], however the notation and complete definition is presented here in the following manner in terms of the Mellin-Barnes type integrals (see also [2]):

$$\aleph[z] = \aleph_{p_i, q_i, \tau_i; r}^{m, n}[Z] = \aleph_{p_i, q_i, \tau_i; r}^{m, n} \left[z \left| \begin{matrix} (a_j, A_j)_{1, n}, [\tau_i(a_{ji}, A_{ji})]_{n+1, p_i; r} \\ (b_j, B_j)_{1, m}, [\tau_i(b_{ji}, B_{ji})]_{m+1, q_i; r} \end{matrix} \right. \right] = \frac{1}{2\pi i} \int_L \Omega_{p_i, q_i, \tau_i; r}^{m, n}(\xi) z^{-\xi} d\xi \tag{1}$$

for all $z \neq 0$, where $i = \sqrt{-1}$ and

$$\Omega_{p_i, q_i, \tau_i; r}^{m, n}(\xi) = \frac{\prod_{j=1}^m \Gamma(b_j + B_j \xi) \prod_{j=1}^n \Gamma(1 - a_j - A_j \xi)}{\sum_{i=1}^r \tau_i \prod_{j=n+1}^{p_i} \Gamma(a_{ji} + A_{ji} \xi) \prod_{j=m+1}^{q_i} \Gamma(1 - b_{ji} - B_{ji} \xi)} \tag{2}$$

the integration path $L = L_{i\gamma\infty}$, $\gamma \in R$, extended from $\gamma - i\infty$ to $\gamma + i\infty$, and is such that the poles, assumed to be simple of $\Gamma(1 - a_j - A_j \xi)$, $j = 1, \dots, n$ do not coincide with the poles of $\Gamma(b_j + B_j \xi)$ $j = 1, \dots, m$ the parameter p_i, q_i are non-negative integers satisfying $0 \leq n \leq p_i, 1 \leq m \leq q_i, \tau > 0$ for $i = 1, \dots, r$. The parameter $A_j, B_j, A_{ji}, B_{ji} > 0$ and $a_j, b_j, a_{ji}, b_{ji} \in C$. The empty product in (2) is interpreted as unity. The existence conditions for the defining integral (1) are given below:

$$\phi_i > 0, |arg(z)| < \frac{\pi}{2} \phi_i, \quad i = 1, \dots, r \tag{3}$$

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$$\phi_i \geq 0, |arg(z)| < \frac{\pi}{2} \phi_i, \text{ and } R(\xi_i) + 1 < 0 \tag{4}$$

where

$$\phi_i = \sum_{j=1}^n A_j + \sum_{j=1}^m B_j - \tau_i \left(\sum_{j=n+1}^{p_i} A_{ji} + \sum_{j=m+1}^{q_i} B_{ji} \right) \tag{5}$$

and

$$\xi_i = \sum_{j=1}^m b_j - \sum_{j=1}^n a_j + \tau_i \left(\sum_{j=m+1}^{q_i} b_{ji} - \sum_{j=n+1}^{p_i} a_{ji} \right) + \frac{1}{2}(p_i - q_i), \quad i = 1, \dots, r \tag{6}$$

for detailed account of the Aleph (\aleph)-function see *Südländ* et al. [1, 2]. For $\tau_i = 1, \forall i = 1, \dots, r$ in (1), we get the I-function defined as follows (see Saxena [3]):

$$I[z] = \aleph_{p_i, q_i, 1; r}^{m, n}[Z] = \aleph_{p_i, q_i, 1; r}^{m, n} \left[z \Big|_{(b_j, B_j)_{1, m}, (b_{ji}, B_{ji})_{m+1, q_i; r}}^{(a_j, A_j)_{1, n}, (a_{ji}, A_{ji})_{n+1, p_i; r}} \right] = \frac{1}{2\pi i} \int_L \Omega_{p_i, q_i, 1; r}^{m, n}(\xi) z^{-\xi} d\xi \tag{7}$$

where the kernel $\Omega_{p_i, q_i, 1; r}^{m, n}(\xi)$ is given in (2). The existence conditions for the integral in (7) are the same as given in (3) through (6) with $\tau_i = 1, i = 1, \dots, r$. If we set $r = 1$, then (7) reduces to the familiar H-function defined as follows (see [4]):

$$H_{p, q}^{m, n}[z] = \aleph_{p_i, q_i, 1; 1}^{m, n}[Z] = \aleph_{p_i, q_i, 1; 1}^{m, n} \left[z \Big|_{(b_q, B_q)}^{(a_p, A_p)} \right] = \frac{1}{2\pi i} \int_L \Omega_{p_i, q_i, 1; 1}^{m, n}(\xi) z^{-\xi} d\xi \tag{8}$$

where the kernel $\Omega_{p_i, q_i, 1; 1}^{m, n}(\xi)$ can be obtained from (2).

The series representation of Aleph (\aleph)-function is given by [5]:

$$\aleph_{p'_i, q'_i, \tau'_i; r'}^{m', n'}[Z] = \aleph_{p'_i, q'_i, \tau'_i; r'}^{m', n'} \left[z \Big|_{(b'_j, B'_j)_{1, m'}, (b'_{ji}, B'_{ji})_{m'+1, q'_i; r'}}^{(a'_j, A'_j)_{1, n'}, (a'_{ji}, A'_{ji})_{n'+1, p'_i; r'}} \right] = \sum_{h=1}^{m'} \sum_{k'=0}^{\infty} \frac{(-1)^{k'} \phi'(s)}{B'_h k'!} z^{-s} \tag{9}$$

where

$$\phi'(s) = \frac{\prod_{j=1}^{m'} \Gamma(b'_j + B'_j s) \prod_{j=1}^{n'} \Gamma(1 - a'_j - A'_j s)}{\prod_{i=1}^{r'} \tau'_i \prod_{j=m'+1}^{q'_i} \Gamma(1 - b'_{ji} - B'_{ji} s) \prod_{j=n'+1}^{p'_i} \Gamma(a'_{ji} + A'_{ji} s)} \tag{10}$$

and

$$s = \eta_{h, k'} = \frac{b'_h + k'}{B'_h}, \quad p'_i < q'_i, |z| < 1.$$

Also, the Fox-Wright's function [6] is defined as

$$p' \Psi_{q'}(z) = p' \Psi_{q'} \left[\begin{matrix} (e_j, E_j)_{1, p'} \\ (f_j, F_j)_{1, q'} \end{matrix}; z \right] = \sum_{l=0}^{\infty} \frac{\prod_{j=1}^{p'} \Gamma(e_j + E_j l)}{\prod_{j=1}^{q'} \Gamma(f_j + F_j l)} \frac{z^l}{l!} \tag{11}$$

where $E_j (j = 1, \dots, p')$ and $F_j (j = 1, \dots, q')$ are real and positive and

$$1 + \sum_{j=1}^{q'} F_j - \sum_{j=1}^{p'} E_j > 0,$$

if we set, some suitable value to the parameters involved in (11) then its reduce into some useful functions as given in [7].

Where, $P_{\beta}^{(a,b)}(z)$ is the Jacobi polynomials ([8], p.68) and is given by

$$P_{\beta}^{(a,b)}(z) = \frac{\Gamma(a+\beta+1)}{\beta!\Gamma(a+b+\beta+1)} \sum_{n'=0}^{[\beta]} \binom{\beta}{n'} \frac{\Gamma(a+b+\beta+n'+1)}{\Gamma(a+n'+1)} \left(\frac{z-1}{2}\right)^{\beta} \quad (12)$$

for more hypergeometric series, see [9].

Plates are the flat structures whose thickness t is small compared to the other in-plane dimension is the radius ρ . Plate theories are classified in many ways. One of them is based on the thickness, that is, thin and thick-plate theories. Geometrically, a plate is said to be thin if its thickness ratio t/ρ is less than $1/20$, otherwise the plate is known to be thick. The bending properties of a plate depend mainly on its thickness as compared with its other dimensions. There are several theories for plates under large deflection; the most commonly used of them is the Von-Karman plate theory which is sometimes referred to as the Kirchoff-Foppel plate theory.

In the classical theory of plates, small deflection and elastic behavior of the material are assumed. When the lateral deflection exceeds one half the plate thickness [10], the classical theory generally is not adequate and the second order effects of the vertical displacements on the membrane stresses need to be considered. Two-coupled non-linear partial differential equations considering these effects were given by [11]. Solutions based on these differential equations have been known as large deflection solutions. Berger [12] in 1955 proposed an approximate method for investigating the large deflection of initially flat isotropic plates. Here the large deflection of a clamped circular plate under non-uniform load has been calculated by using Berger's approximate method. We consider the applied external pressure $p(x)$ in the following form:

$$P(x) = K_0 \left(1 - \frac{x^2}{\rho^2}\right)^{\alpha} P_{\beta}^{a,b} \left(1 - \frac{2x^2}{\rho^2}\right) {}_{p',q'}\Psi_{q'} \left\{ K_1 \left(1 - \frac{x^2}{\rho^2}\right) \right\} \aleph_{p_i, q_i, \tau_i; r}^{m, n} \left[K_2 \left(1 - \frac{x^2}{\rho^2}\right) \right] \aleph_{p'_i, q'_i, \tau'_i; r'}^{m', n'} \left[K_3 \left(1 - \frac{x^2}{\rho^2}\right) \right] \quad (13)$$

where K_0 , K_1 , K_2 and K_3 are constants. Recently, some research work based on modelling had done in [13].

2. Statement of the problem

Let us assume a clamped circular plate of thickness t , radius ρ and flexural rigidity R . Then by using Berger's method, the approximate equations for a circular plate undergoing large deflections due to an externally applied load $p(x)$ may be given as

$$\left(\frac{d^2}{dx^2} + \frac{1}{x} \frac{d}{dx}\right) \left(\frac{d^2 w}{dx^2} + \frac{1}{x} \frac{dw}{dx} - k^2 w\right) = \frac{P}{R} = \phi(x) \quad (14)$$

where k is a normalized constant of integration given by the equation

$$\frac{dy}{dx} + \frac{y}{x} + \frac{1}{2} \left(\frac{dw}{dx}\right)^2 = \frac{k^2 t^2}{12} \quad (15)$$

where w is the plate deflection, normal to the middle plane of the plate and y is the radial displacement. The boundary condition of the problem are:

- (i) $w = 0 = \frac{dw}{dx}$, at $x = \rho$
- (ii) $y = 0$, at $x = \rho$

Solution of the problem

Let us consider

$$w = \sum_i G_i [J_0(x t_i) - J_0(\rho t_i)] \quad (16)$$

where t_i is the i th root of $J_1(\rho t_i) = 0$.

It is clear that boundary conditions are satisfied by the above equation. Now using (16) in the Eq. (14), we find

$$\sum_i G_i t_i^2 (k^2 + t_i^2) J_0(x t_i) = \phi(x) \quad (17)$$

now expanding $\phi(x)$ in a series of Bessel's function, we obtain on integration

$$\int_0^{\rho} G_i t_i^2 (k^2 + t_i^2) J_0^2(x t_i) x dx = \int_0^{\rho} \phi(x) J_0(x t_i) \rho dx \quad (18)$$

now by left hand side of (18)

$$\int_0^\rho x J_0^2(x t_i) dx = \frac{\rho^2}{2} J_0^2(\rho t_i) \tag{19}$$

and (18) becomes

$$G_i t_i^2 (k^2 + t_i^2) \frac{\rho^2}{2} J_0^2(\rho t_i) = \int_0^\rho \phi(x) J_0(x t_i) x dx$$

or

$$G_i = \frac{2 \int_0^\rho x \phi(x) J_0(x t_i) dx}{\rho^2 t_i^2 (k^2 + t_i^2) J_0^2(\rho t_i)} \tag{20}$$

Now using [14], Eqs. (2), (9) and (10), the definition of Bessel function and interchanging the order of summations and integration, we find

$$\begin{aligned} & \int_0^1 \theta^{2\lambda+1} (1-\theta^2)^\alpha P_\beta^{a,b} (1-2\theta^2)_{p'} \psi_{q'} [K_1(1-\theta^2)] \mathfrak{N}_{p_i, q_i, \tau_i; r}^{m, n} [K_2(1-\theta^2)] \mathfrak{N}_{p'_i, q'_i, \tau'_i; r'}^{m', n'} [K_3(1-\theta^2)] J_\mu(\theta \tau) d\theta \\ &= \sum_{l=0}^\infty \sum_{n'=0}^\beta \sum_{n''=0}^\infty \sum_{h=1}^{m'} \sum_{k'=0}^\infty \frac{K_1^l K_3^{-s} (-1)^{k'+n''} (-\beta)_{n'} \phi'(s) (\frac{\tau}{2})^{\mu+2n''}}{2! n'! n''! k'! B'_h} \\ & \cdot \frac{\prod_{j=1}^{p'} \Gamma(e_j + E_j l) \Gamma(1+a+\beta) (1+a+b+\beta)_{n'} \Gamma(\lambda+n'+n'' + \frac{\mu}{2} + 1)}{\prod_{j=1}^{q'} \Gamma(f_j + F_j n) \Gamma(1+a+n') \Gamma(1+\mu+n'')} \\ & \cdot \mathfrak{N}_{p_i+1, q_i+1, \tau_i; r}^{m, n+1} [K_2 \left| \begin{matrix} (-\alpha-l+s, 1), (a_j, A_j)_{1, n}, [\tau_i(a_{ji}, A_{ji})]_{n+1, p_i; r} \\ (b_j, B_j)_{1, m}, [\tau_i(b_{ji}, B_{ji})]_{m+1, q_i; r}, (-1-\lambda-l-n'-n''-\alpha+s, 1) \end{matrix} \right|] \end{aligned} \tag{21}$$

where

$$Re(a) > -1, Re(b) > -1, Re(\lambda) > -1, Re(\alpha) > -1,$$

$$Re(\mu) > -\frac{1}{2}, Re(\alpha + \frac{b_j}{B_j}) > 0, (j = 1, \dots, q_i; \text{ here } i = 1, \dots, r)$$

and

$$Re(\alpha + \frac{b'_j}{B'_j}) > 0, (j = 1, \dots, q'_i; \text{ here } i = 1, \dots, r')$$

Using (21) in view of (13) and (14), we get

$$\begin{aligned} G_i &= \frac{K_0 \Gamma(1+a+\beta)}{R t_i^2 \beta! (k^2 + t_i^2) J_0^2(\rho t_i)} \sum_{l=0}^\infty \sum_{n'=0}^\beta \sum_{n''=0}^\infty \sum_{h=1}^{m'} \sum_{k'=0}^\infty \frac{K_1^l K_3^{-s} (-1)^{k'+n''} (-\beta)_{n'} \phi'(s)}{l! n'! n''! k'! B'_h} \phi'(s) \\ & \cdot \frac{\prod_{j=1}^{p'} \Gamma(e_j + E_j l) \Gamma(1+n'+n'') (1+a+b+\beta)_{n'}}{\prod_{j=1}^{q'} \Gamma(f_j + F_j l) \Gamma(1+a+n') \Gamma(1+n'')} \\ & \cdot \mathfrak{N}_{p_i+1, q_i+1, \tau_i; r}^{m, n+1} [K_2 \left| \begin{matrix} (-\alpha-l+s, 1), (a_j, A_j)_{l, n}, [\tau_i(a_{ji}, A_{ji})]_{n+1, p_i; r} \\ (b_j, B_j)_{l, m}, [\tau_i(b_{ji}, B_{ji})]_{m+1, q_i; r}, (-1-l-n'-n''-\alpha+s, 1) \end{matrix} \right|]. \end{aligned} \tag{22}$$

Now combining the Eqs. (16) and (22), we get

$$w = L_1 \sum_i \frac{L_2}{k^2 + t_i^2} [J_0(x t_i) - J_0(\rho t_i)] \quad (23)$$

where

$$L_1 = \frac{K_0 \Gamma(1 + a + \beta)}{R \beta!}$$

and

$$L_2 = \frac{1}{t_i^2 J_0^2(\rho t_i)} \sum_{l=0}^{\infty} \sum_{n'=0}^{\beta} \sum_{n''=0}^{\infty} \sum_{h=1}^{m'} \sum_{k'=0}^{\infty} \frac{K_1^l K_3^{-s} (-1)^{k'+n''} (-\beta)_{n'}}{l! n'! n''! k'! B_h'} \phi'(s) \cdot \frac{\prod_{j=1}^{p'} \Gamma(e_j + E_j l) \Gamma(1 + n' + n'') (1 + a + b + \beta)_{n'}}{\prod_{j=1}^{q'} \Gamma(f_j + F_j l) \Gamma(1 + a + n') \Gamma(1 + n'')} \cdot \mathfrak{K}_{p_i+1, q_i+1, \tau_i; r}^{m, n+1} [K_2 \left| \begin{matrix} (-\alpha-l+s, 1), (a_j, A_j)_{l, n}, [\tau_i(a_{ji}, A_{ji})]_{n+1, p_i; r} \\ (b_j, B_j)_{l, m}, [\tau_i(b_{ji}, B_{ji})]_{m+1, q_i; r}, (-1-l-n'-n''-\alpha+s, 1) \end{matrix} \right|].$$

Now the radial displacement y can be obtained by using Eqs. (15) and (16) as

$$y = \frac{k^2 t^2 x}{24} - \frac{1}{2} \sum_{i=1}^{\infty} G_i^2 t_i^2 \left[\frac{x}{2} \left\{ J_1'(x t_i) + \left(1 - \frac{1}{x^2 t_i^2} \right) J_1^2(x t_i) \right\} \right] - \frac{1}{2} \sum_{i=1}^{\infty} \sum_{j=1}^{\infty} G_i G_j t_i t_j \left[\frac{t_i J_2(x t_i) J_1(x t_j) - t_j J_2(x t_j) J_1(x t_i)}{t_i^2 - t_j^2} \right] + C_1, i \neq j \quad (24)$$

where C_1 is the constant of integration.

Applying the boundary condition $y=0$ at $x = \rho$ and $J_1(\rho t_i) = 0$, we get

$$C_1 = -\frac{k^2 t^2 \rho}{24} + \frac{1}{4} \sum_{i=1}^{\infty} G_i^2 t_i^2 \rho J_1^2(\rho t_i) \quad (25)$$

hence the radial displacement y is established as

$$y = \frac{k^2 t^2 (x - \rho)}{24} - \frac{1}{2} \sum_{i=1}^{\infty} G_i^2 t_i^2 \left[\frac{x}{2} \left\{ J_1'(x t_i) + \left(1 - \frac{1}{x^2 t_i^2} \right) J_1^2(x t_i) \right\} \right] - \frac{1}{2} \sum_{i=1}^{\infty} \sum_{j=1}^{\infty} G_i G_j t_i t_j \left[\frac{t_i J_2(x t_i) J_1(x t_j) - t_j J_2(x t_j) J_1(x t_i)}{t_i^2 - t_j^2} \right] + \frac{1}{4} \sum_{i=1}^{\infty} G_i^2 t_i^2 \rho J_0^2(\rho t_i).$$

3. Applications

(3.A) The deflection given by Eq. (23) can be used to evaluate the boundary stresses at the surface of the plate which for the circular plate, are given by [12] as

$$\sigma_x = \frac{-6R}{t^2} \left(\frac{d^2 w}{dx^2} + \frac{\nu dw}{x dx} \right) \quad (26)$$

and

$$\sigma_\theta = \frac{-6R}{t^2} \left(\nu \frac{d^2 w}{dx^2} + \frac{1 dw}{x dx} \right) \quad (27)$$

where ν is the Poisson's ratio.

By using (23), we get

$$\sigma_x = \frac{-6R}{t^2} L_1 \sum_i \frac{L_2}{(k^2 + t_i^2)} \left[J_0''(xt_i) + \frac{v}{x} J_0'(xt_i) \right] \tag{28}$$

and

$$\sigma_\theta = \frac{-6R}{t^2} L_1 \sum_i \frac{L_2}{(k^2 + t_i^2)} \left[v J_0''(xt_i) + \frac{1}{x} J_0'(xt_i) \right] \tag{29}$$

Now, putting $x = 0$ in (28) and (29), we get the bending stresses at the centre of the plate as

$$(\sigma_x)_{x=0} = (\sigma_\theta)_{x=0} = \frac{3R}{t^2} L_1 \sum_i \frac{L_2}{(k^2 + t_i^2)} (v + 1) t_i^2 \tag{30}$$

also by putting $x = \rho$, the bending stresses at the edge of the plate are obtained as

$$(\sigma_x)_{x=\rho} = \frac{6R}{t^2} L_1 \sum_i \frac{L_2}{(k^2 + t_i^2)} t_i^2 J_0(\rho t_i) \tag{31}$$

and

$$(\sigma_\theta)_{x=\rho} = \frac{6R}{t^2} L_1 \sum_i \frac{L_2}{(k^2 + t_i^2)} v t_i^2 J_0(\rho t_i) \tag{32}$$

(3.B) When $k = 0$, the differential Eq. (14) corresponds to that of small deflection equation and then Eq. (23) leads to

$$w = L_1 \sum_i \frac{L_2}{t_i^2} [J_0(xt_i) - J_0(\rho t_i)] \tag{33}$$

(3.C) By using $x = 0$, we obtain the deflection w_0 at the centre of the plate as

$$w_0 = L_1 \sum_i \frac{L_2}{(k^2 + t_i^2)} [1 - J_0(\rho t_i)] \tag{34}$$

whereas the small deflection will be given by

$$w_0 = L_1 \sum_i \frac{L_2}{t_i^2} [1 - J_0(\rho t_i)] \tag{35}$$

4. Special cases

(i) By taking $\tau_i = 1 \forall i = 1, \dots, r$ and $\tau'_i = 1 \forall i = 1, \dots, r'$ for $\aleph_{p_i, q_i, \tau_i; r}^{m, n}$ and $\aleph_{p'_i, q'_i, \tau'_i; r'}^{m', n'}$ in the load $p(x)$, both the Aleph-functions reduces to the I-function [3]. Then we obtain the deflection as

$$w = D_1 \sum_i \frac{D_2}{(k^2 + t_i^2)} [J_0(xt_i) - J_0(\rho t_i)] \tag{36}$$

where

$$D_1 = \frac{K_0 \Gamma(1 + a + \beta)}{R \beta!}$$

and

$$D_2 = \frac{1}{t_i^2 J_0^2(\rho t_i)} \sum_{l=0}^{\infty} \sum_{n'=0}^{\beta} \sum_{n''=0}^{\infty} \sum_{h=1}^{m'} \sum_{k'=0}^{\infty} \frac{K_1^l K_3^{-s} (-1)^{k'+n''} (-\beta)_n'}{l! n'! n''! k'! B_h'} \phi'(s') \left(\frac{\rho t_i}{2}\right)^{2n''} \frac{\prod_{j=1}^{p'} \Gamma(e_j + E_j l) \Gamma(1 + n' + n'') (1 + a + b + \beta)_{n'}}{\prod_{j=1}^{q'} \Gamma(f_j + F_j l) \Gamma(1 + a + n') \Gamma(1 + n'')} \cdot I_{p_i+1, q_i+1, 1; r}^{m, n+1} [K_2 |_{(b_j, B_j)_{l, m}, (b_{j_i}, B_{j_i})_{m+1, q_i; r}, (-1-l-n'-n''-\alpha+s', 1)}]$$

whereas we get the small deflection as

$$w = D_1 \sum_i \frac{D_2}{t_i^2} [J_0(xt_i) - J_0(\rho t_i)] \quad (37)$$

in this case, the deflection at the centre of the plate is given by

$$w_0 = D_1 \sum_i \frac{D_2}{(k^2 + t_i^2)} [1 - J_0(\rho t_i)]. \quad (38)$$

(ii) By taking $\tau_i = 1 \forall i = 1, \dots, r$, set $r = 1$ and $\tau'_i = 1 \forall i = 1, \dots, r'$ set $r' = 1$ for $\aleph_{p_i, q_i, \tau_i; r}^{m, n}$ and $\aleph_{p'_i, q'_i, \tau'_i; r'}^{m', n'}$ in the load $p(x)$, both the Aleph-functions reduces to the familiar H-function [4]. Then we reduces to known result obtained by Chaurasia and Arya [15].

Inview of the generality of the \aleph -function, on specializing the various parameters, we can obtain from our results, several results involving a remarkably wide variety of useful functions, which are expressible in terms of the Mittag-Leffler function ([4], p.25, Eq. (1.137)), the generalized Wright hypergeometric function ([4], p.25, Eq. (1.140)), the generalized Bessel-Maitland function ([4], p.25, Eq.(1.139)) and their various special cases. Thus, the results presented in this paper would at once yield a very large number of results involving a large variety of special functions occurring in the problems of science, engineering and mathematical physics etc.

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